

# Heat Exchange Technology Explained

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# Steam Heaters – The New Generation – But what are they?, how do they work?

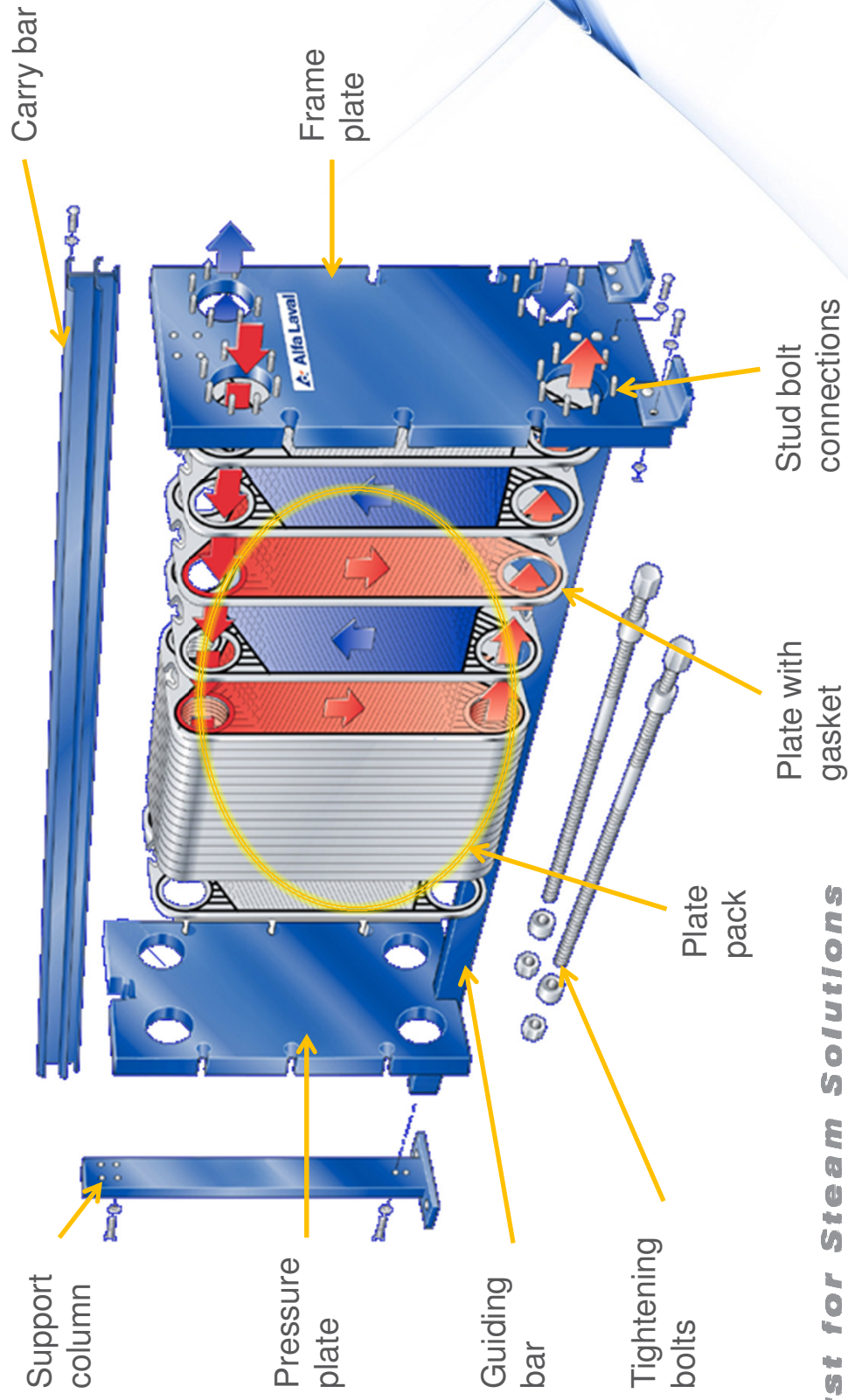


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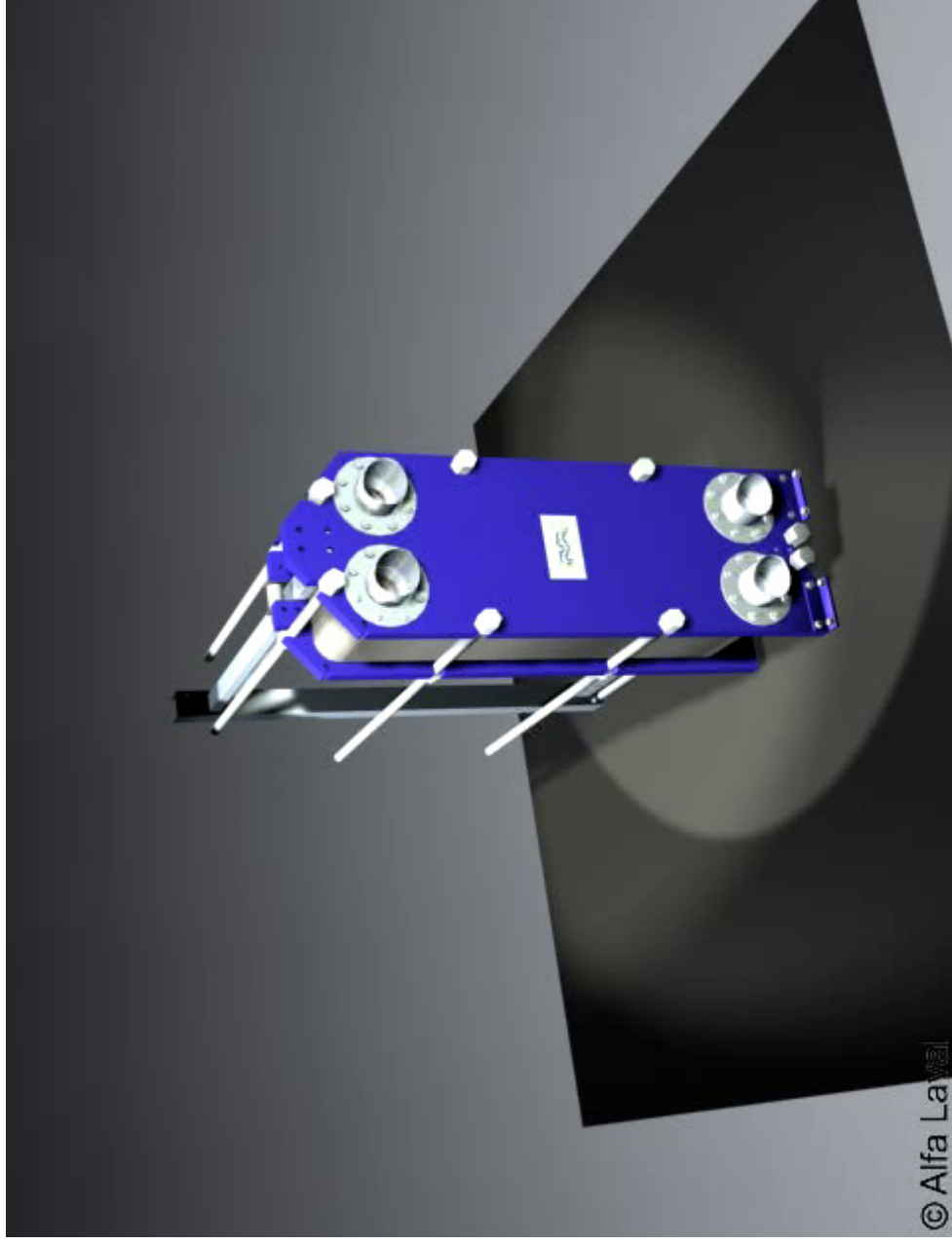
# Plate Heat Exchanger components



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# Plate Heat Exchange flow path video



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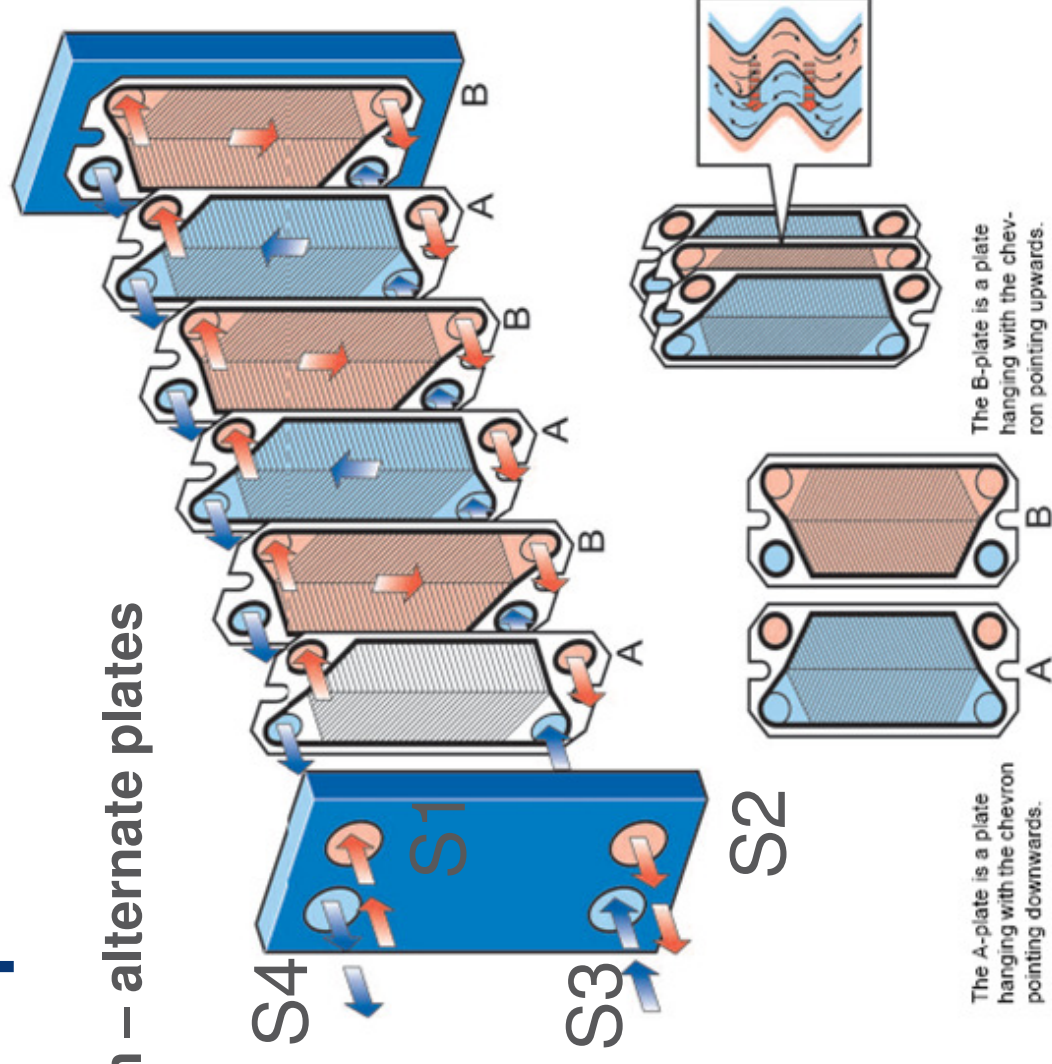
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# Plate flow pattern

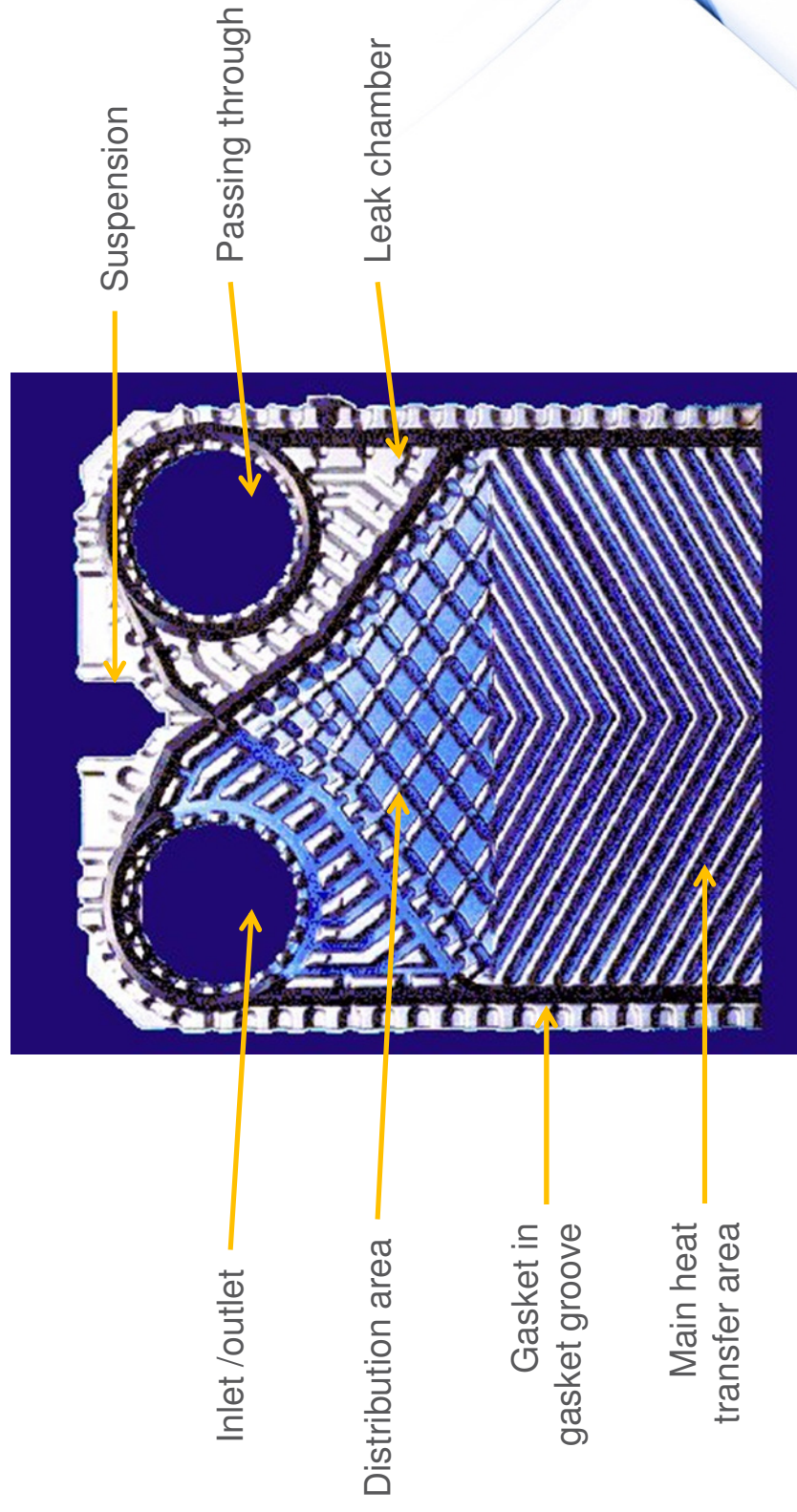
Flow pattern – alternate plates



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# Plate – main components



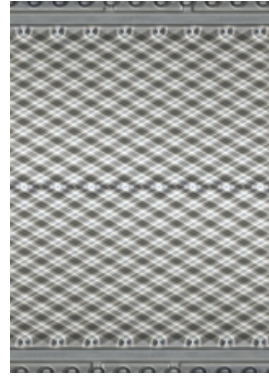
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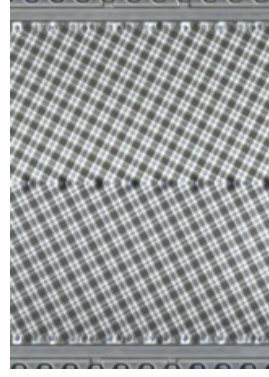
# Plate – corrugation and channels

Low turbulence  
and pressure drop



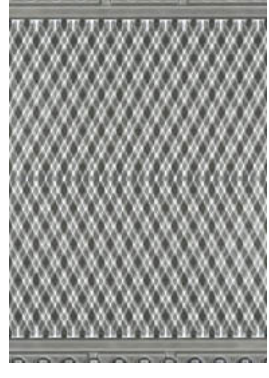
$L + L = L$  channels

Medium turbulence  
and pressure drop



$L + H = M$  channels

High turbulence and  
pressure drop



$H + H = H$  channels



## Advantages

- Efficient heat transfer
- Strong construction

## Benefits

- Low fouling
- Optimal design
- Unaffected by vibration

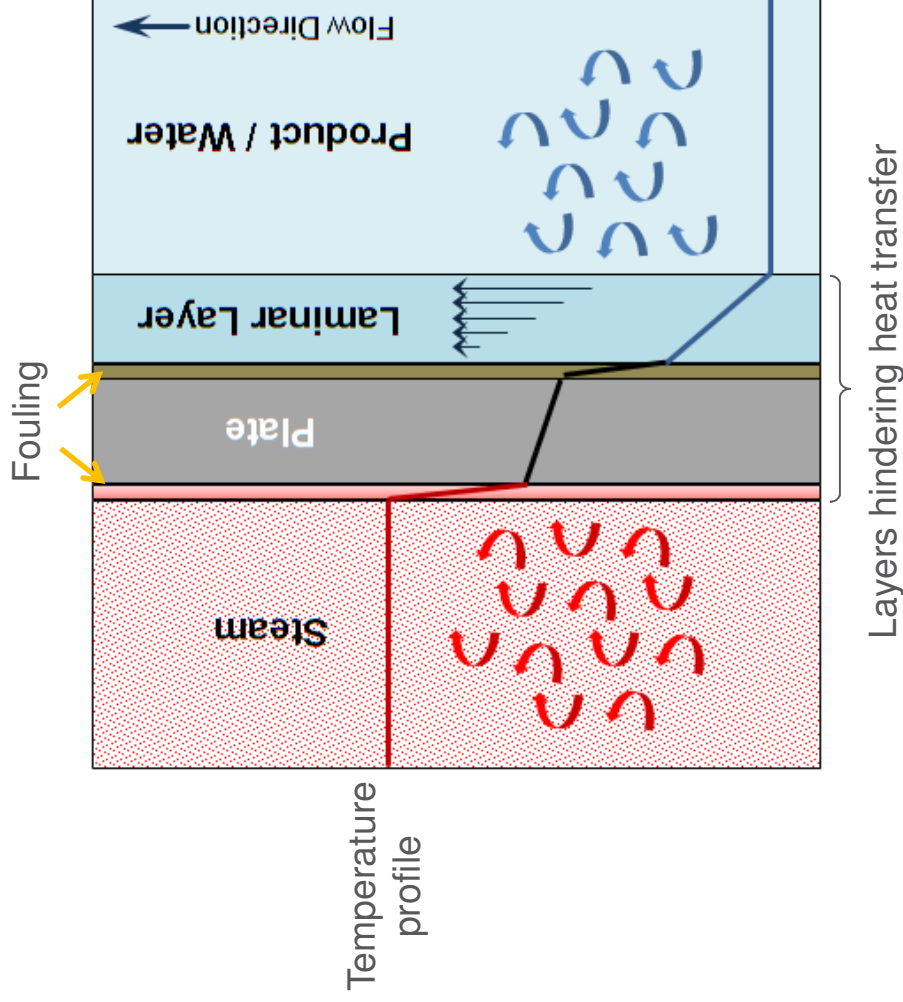
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# Heat Transfer rate

## Increased turbulence aiding heat transfer



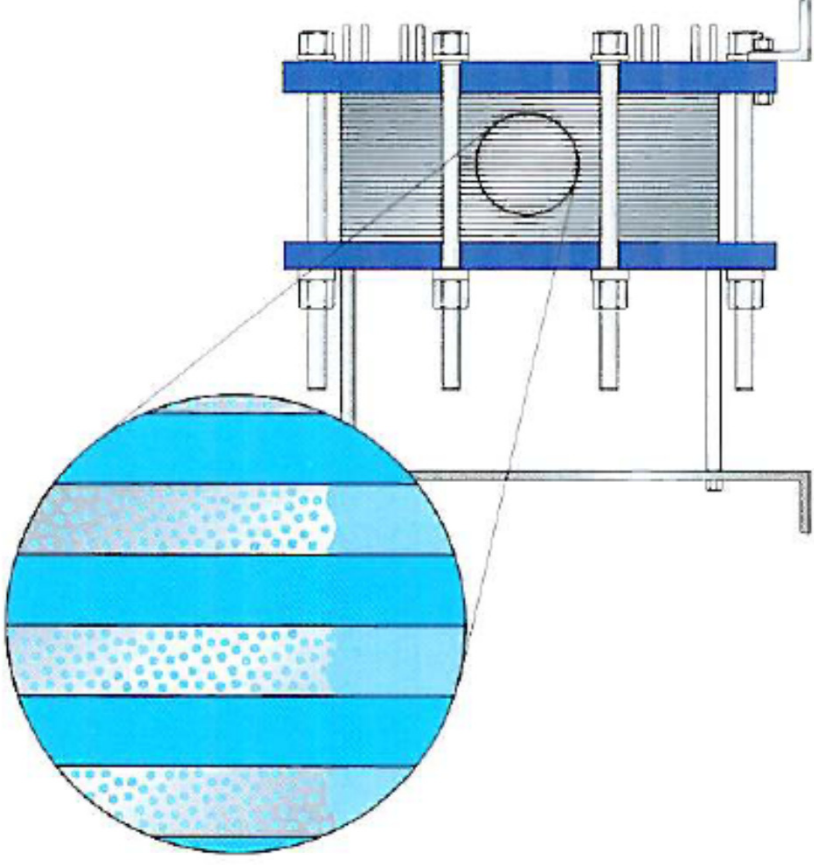
Increased turbulence helps limit the poor heat transfer layers:

- Laminar flow – heat transfer through conduction
- Fouling
  - Scaling
  - Major debris
  - Biological growth
  - Sedimentation
  - Burn-on

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# Low volume and PV ratio aids sub cool in one single pass



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# Heat Transfer -a bit of theory

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# What should the Heat Exchange control achieve?

- Provide the required degree of accuracy
- Be reactive to the secondary load requirement
- Be safe
- Make full use of the energy available in the steam
- **Avoid energy wastage**

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# Heat Transfer equation

This drives the whole process

The diagram features the equation  $Q = A \times k \times LMTD$  in a large, black, serif font. Surrounding the equation are four grey speech bubbles, each containing a variable name in purple text. The bubbles are positioned as follows: 'energy required' is to the left of 'Q'; 'heat exchange area' is to the left of 'A'; 'heat transfer coefficient' is to the right of 'k'; and 'logarithmic mean temperature difference' is to the right of 'LMTD'.

$$Q = A \times k \times LMTD$$

energy required

heat exchange area

heat transfer coefficient

logarithmic mean temperature difference

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# Logarithmic mean temperature difference

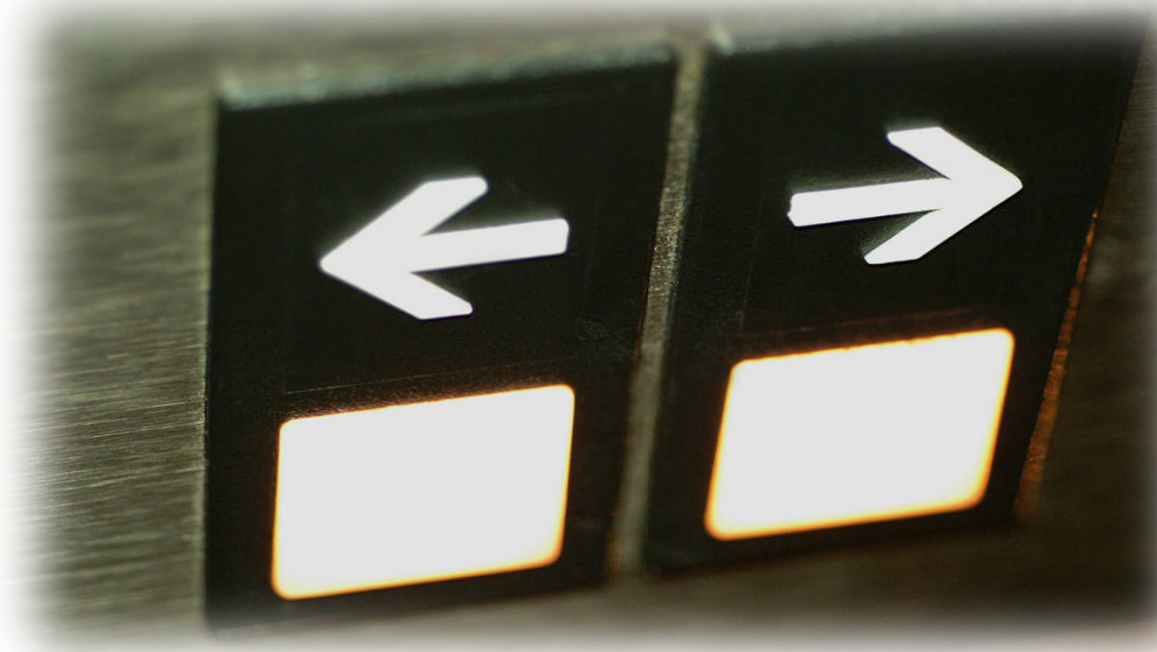
The diagram illustrates the LMTD formula with four callouts identifying the variables:

- primary side (steam) temperature in**: points to  $th_1$
- primary side (steam) temperature out**: points to  $tc_2$
- secondary side temperature in**: points to  $th_2$
- secondary side temperature out**: points to  $tc_1$

$$LMTD = \frac{(th_1 - tc_2) - (th_2 - tc_1)}{\ln \left( \frac{th_1 - tc_2}{th_2 - tc_1} \right)}$$

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# So how do we control - control actions

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# Control actions

$$Q = A \times k \times LMTD$$



the energy  
requirement  
decreases



the heating  
area needs to  
decrease

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# Control actions

or

$$Q = A \times k \times LMTD$$



the energy  
requirement  
decreases



the logarithmic  
mean  
temperature  
needs to  
decrease

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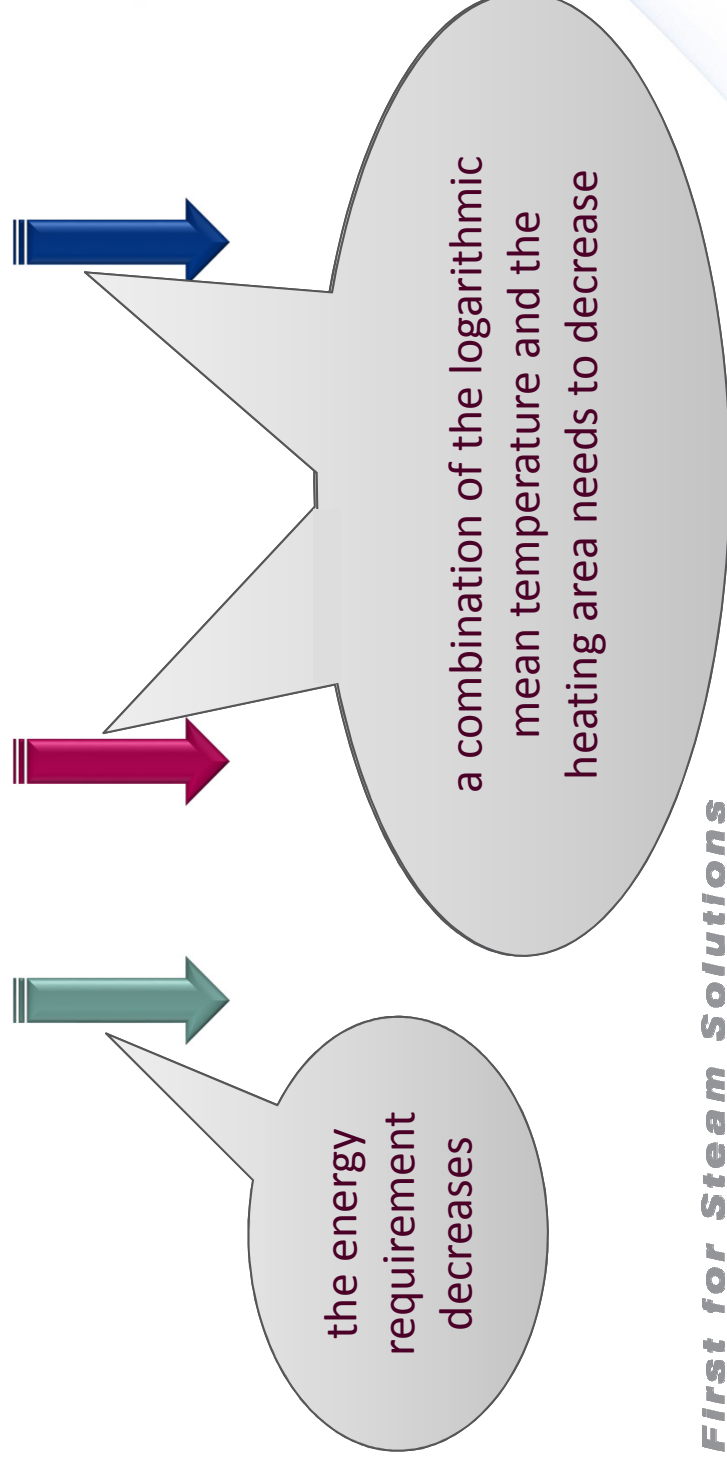
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# Control actions

or

$$Q = A \times k \times LMTD$$



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# Lets review one control method: condensate control

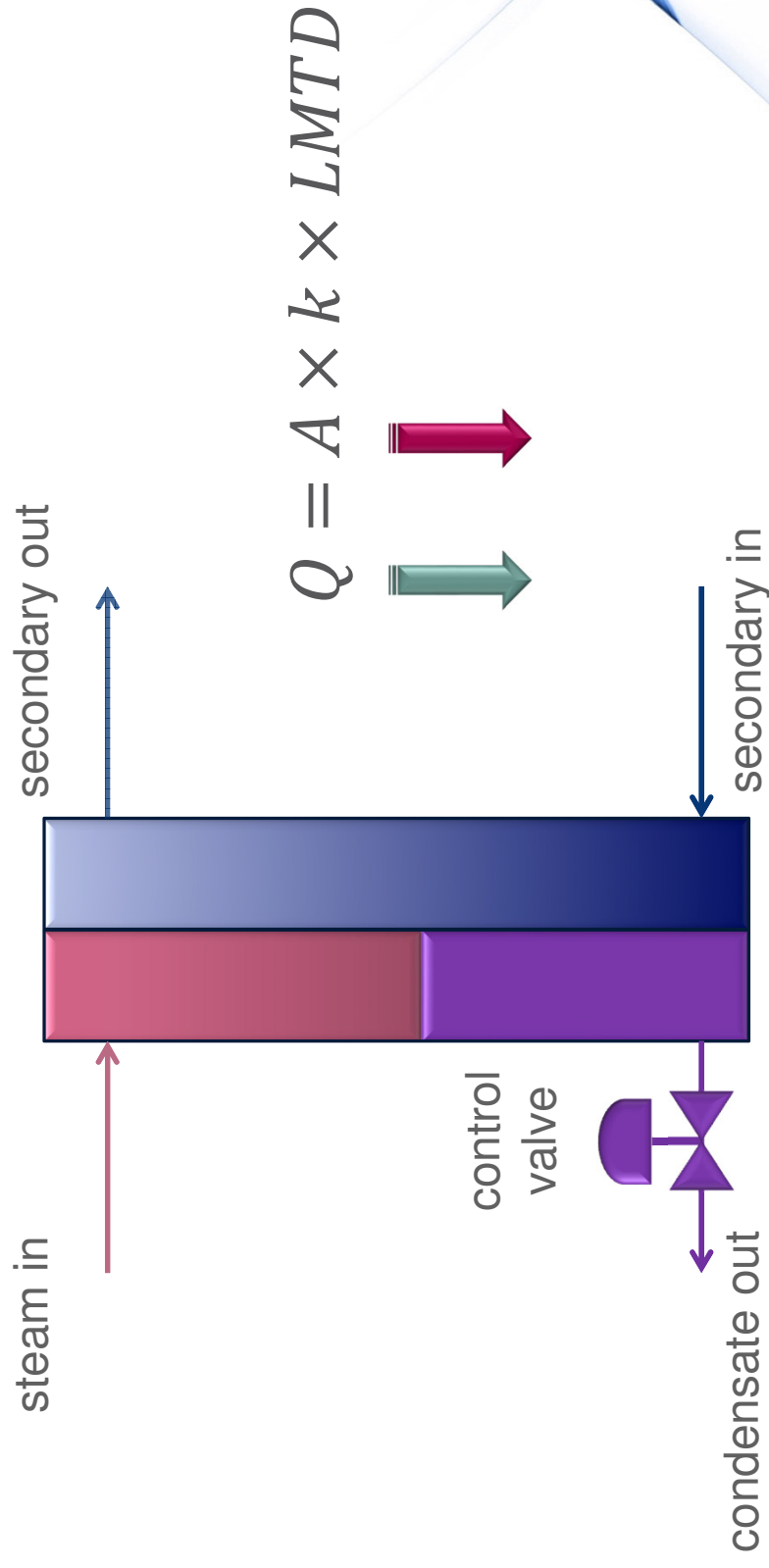
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# Basic control methods

## Control valve on the condensate outlet

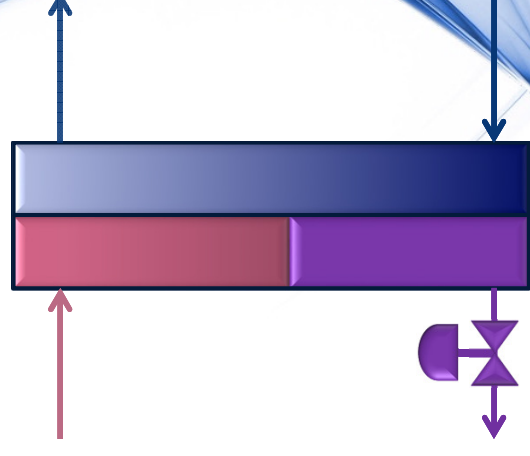


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# Control valve on the condensate outlet


- Control by condensate varies the heat exchange area
- Reacts quite quickly to an increasing load
- React more slowly to a decreasing load
- Varying heat exchanger area varies sub cooling
- When the condensate is sub-cooled, sensible heat pre-heats the secondary side. This in turn:
  - lowers the amount of steam required by the heating process
  - Ensures no flash steam is formed, avoiding a possible source of wasted energy



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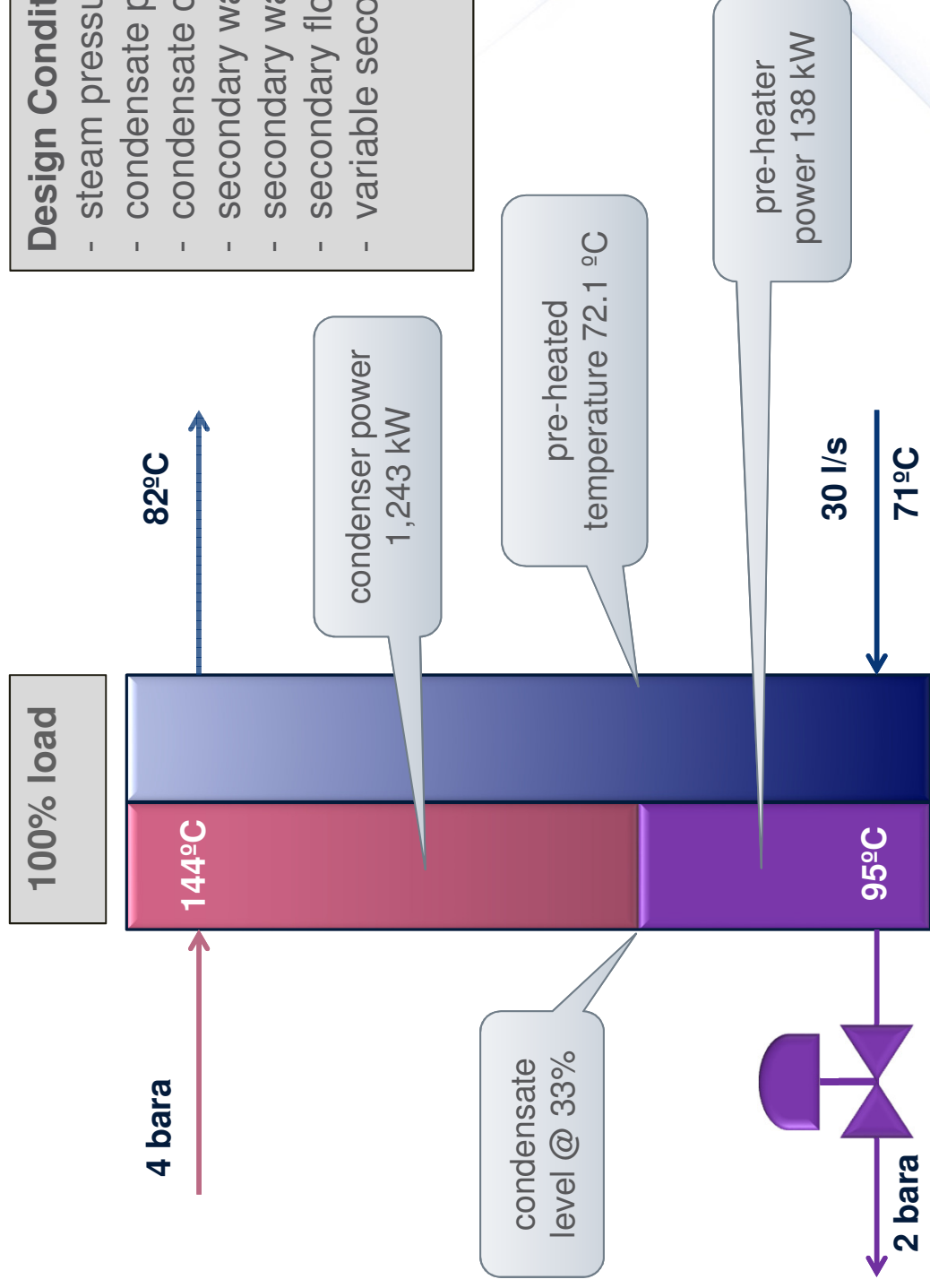
So, how  
does  
it work, what  
does it  
achieve?

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# Control valve on the condensate outlet



## Design Condition

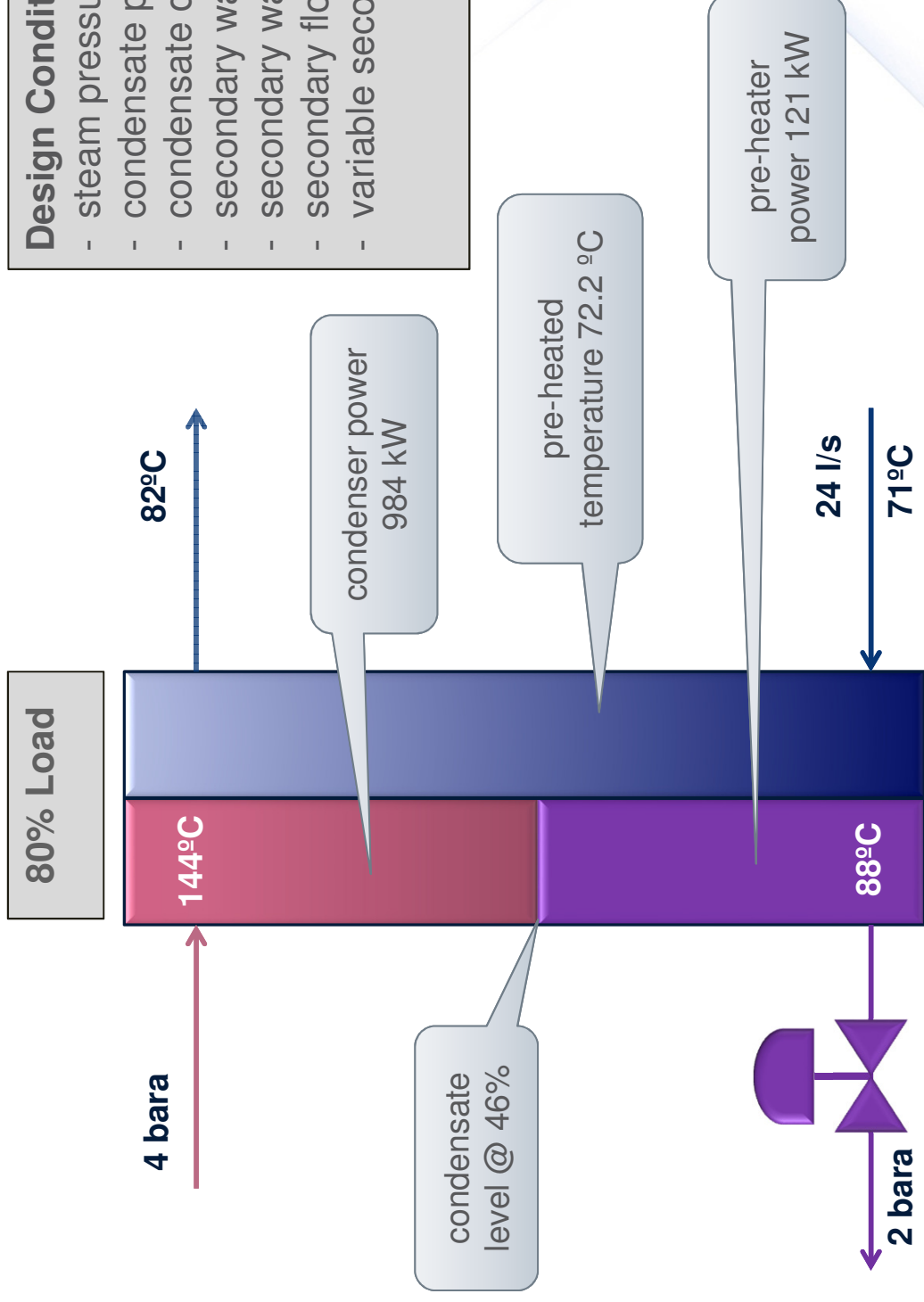
- steam pressure 4 bara
- condensate pressure 2 bara
- condensate discharge 95°C
- secondary water in 71°C
- secondary water out 82°C
- secondary flow 30 l/s
- variable secondary flow

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# Control valve on the condensate outlet



## Design Condition

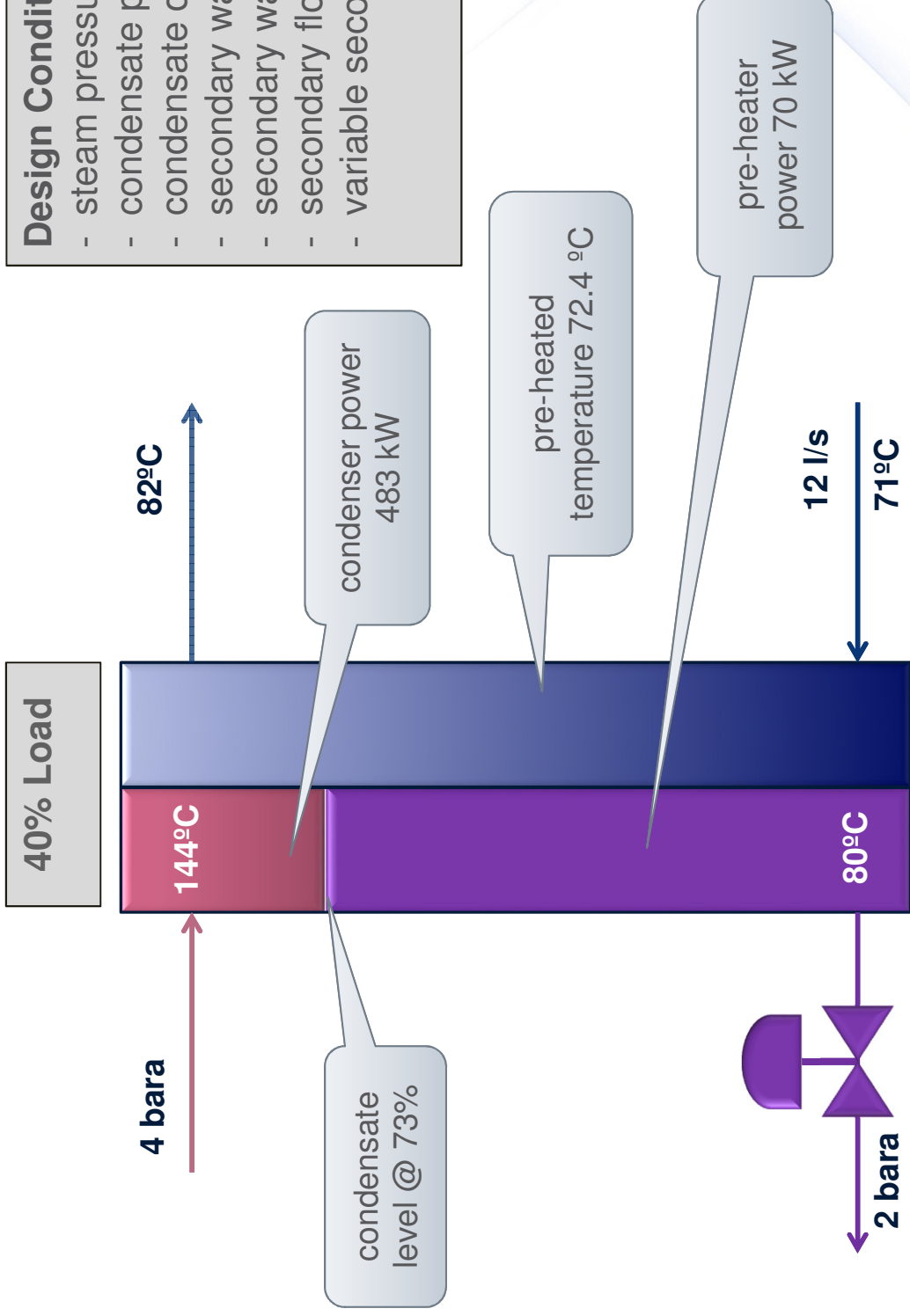
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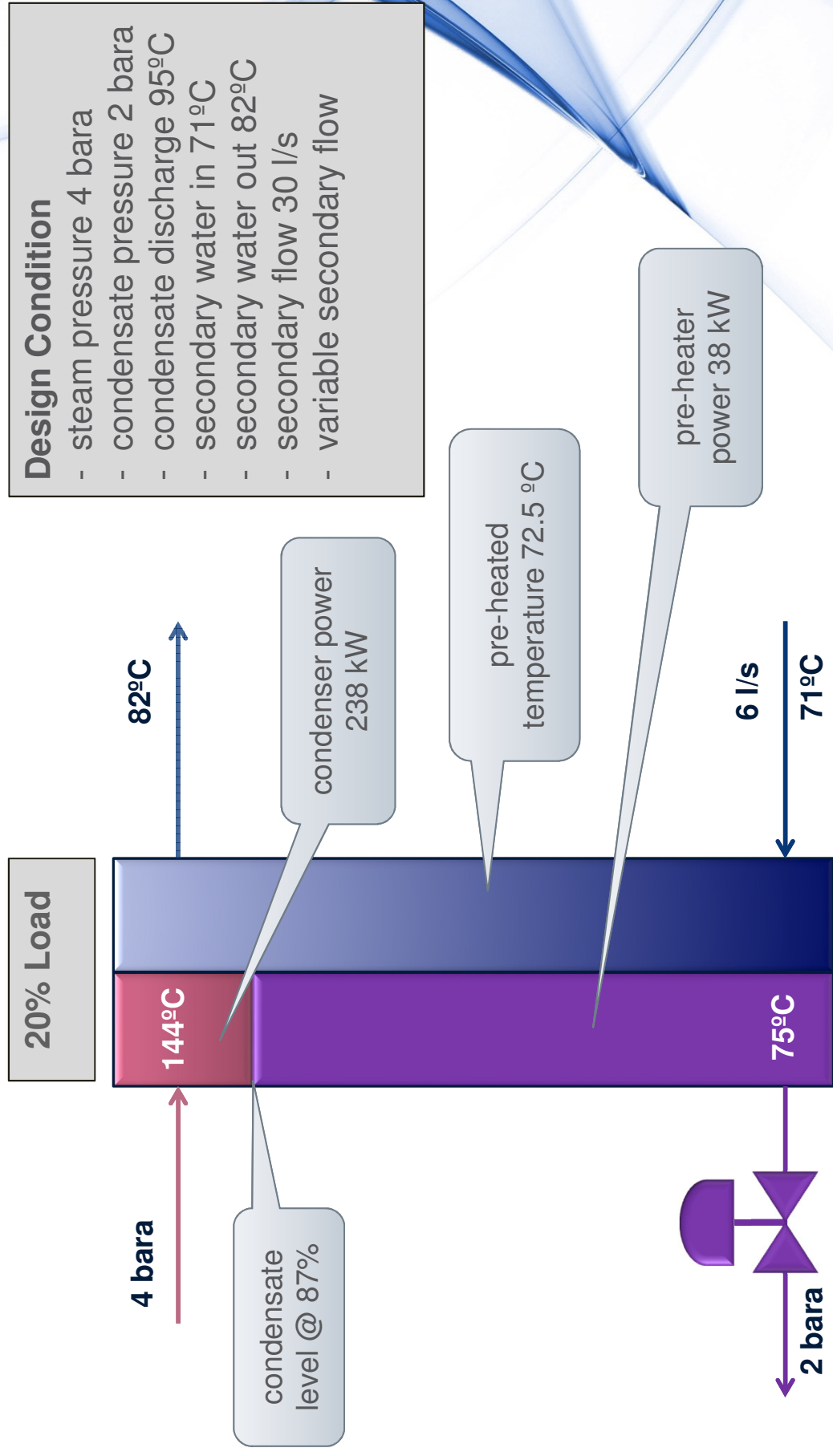
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# Energy savings

- Existing heat exchanger with steam control (LTHW)
- 5 bar g steam supply
- Heat exchanger sized to condense at 2.5 bar g steam
- On substantial load, i.e. 50% year, design KW rating 400KW
- **Heat exchanger sub cools condensate to at least 95°C (full load), but general this will be within 2 degrees of the return temperature**

$$\text{Saving} = (139-95) \times 4.186 \times (400/2153) = 34\text{KW} = 8.5\%$$

Or simply 2.5 bar g to zero is 7.5% and 100 to 95°C is 1%

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Other Benefits...

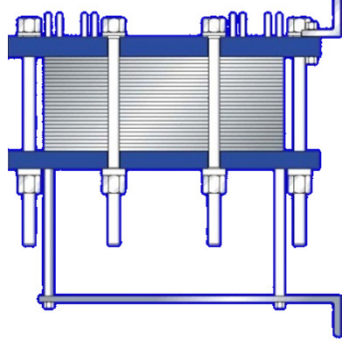
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# PHE benefits compared to shell and tube

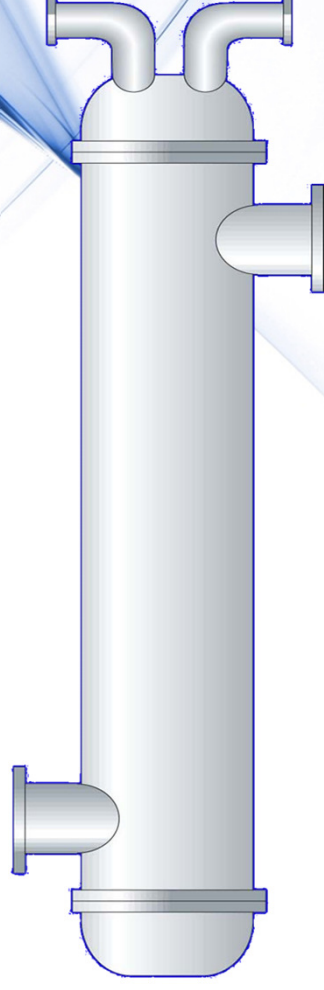
- Excellent temperature control characteristics
- Small  $P \times V$  product => no need for inspection from pressure vessel authorities (splash guards must be fitted)

Small hold-up volume



- Slow response to load changes
- Large  $P \times V$  product => inspection required from pressure vessel authorities

Large hold-up volume



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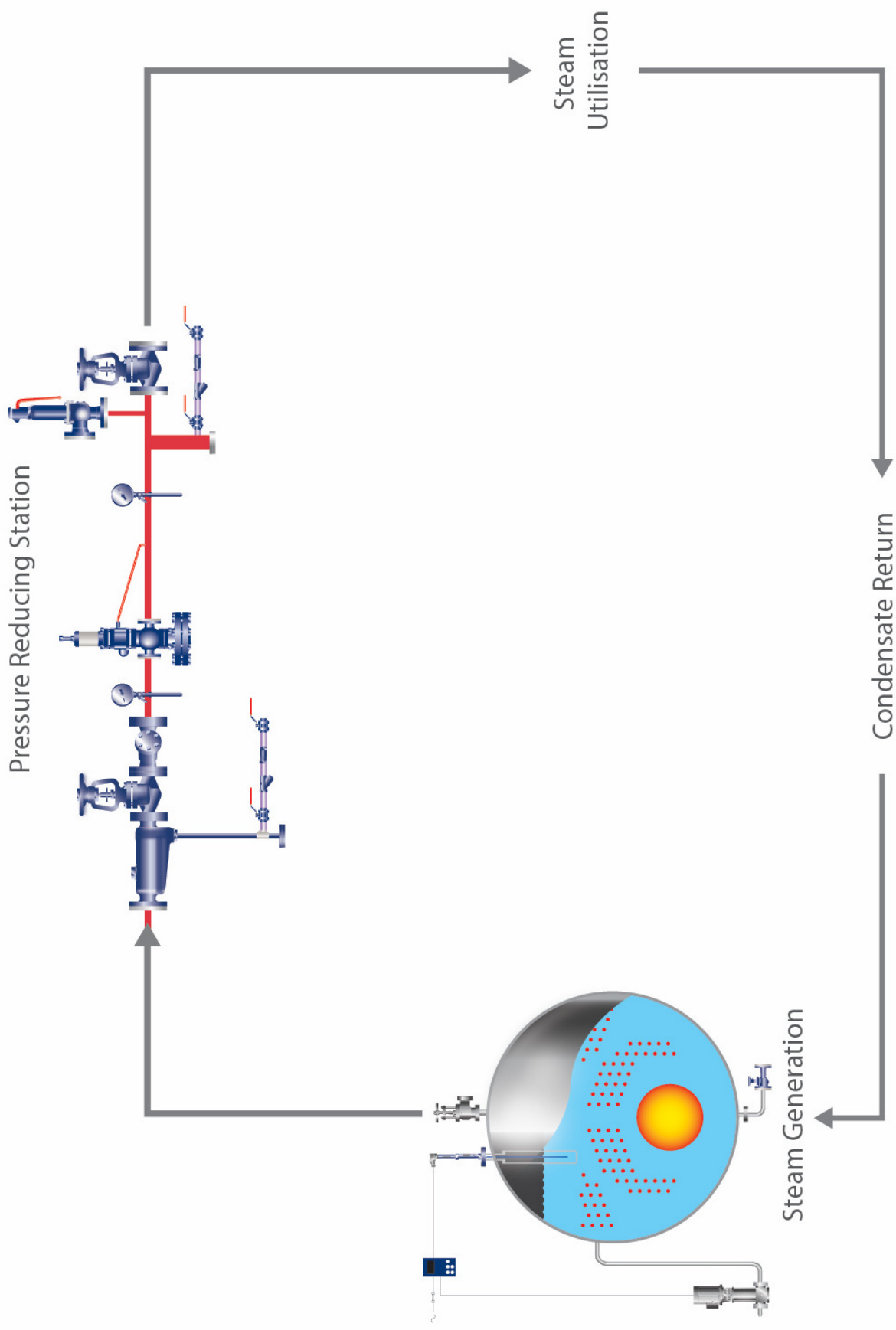
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Add Power Generation to get  
even more efficiency and  
carbon reduction...

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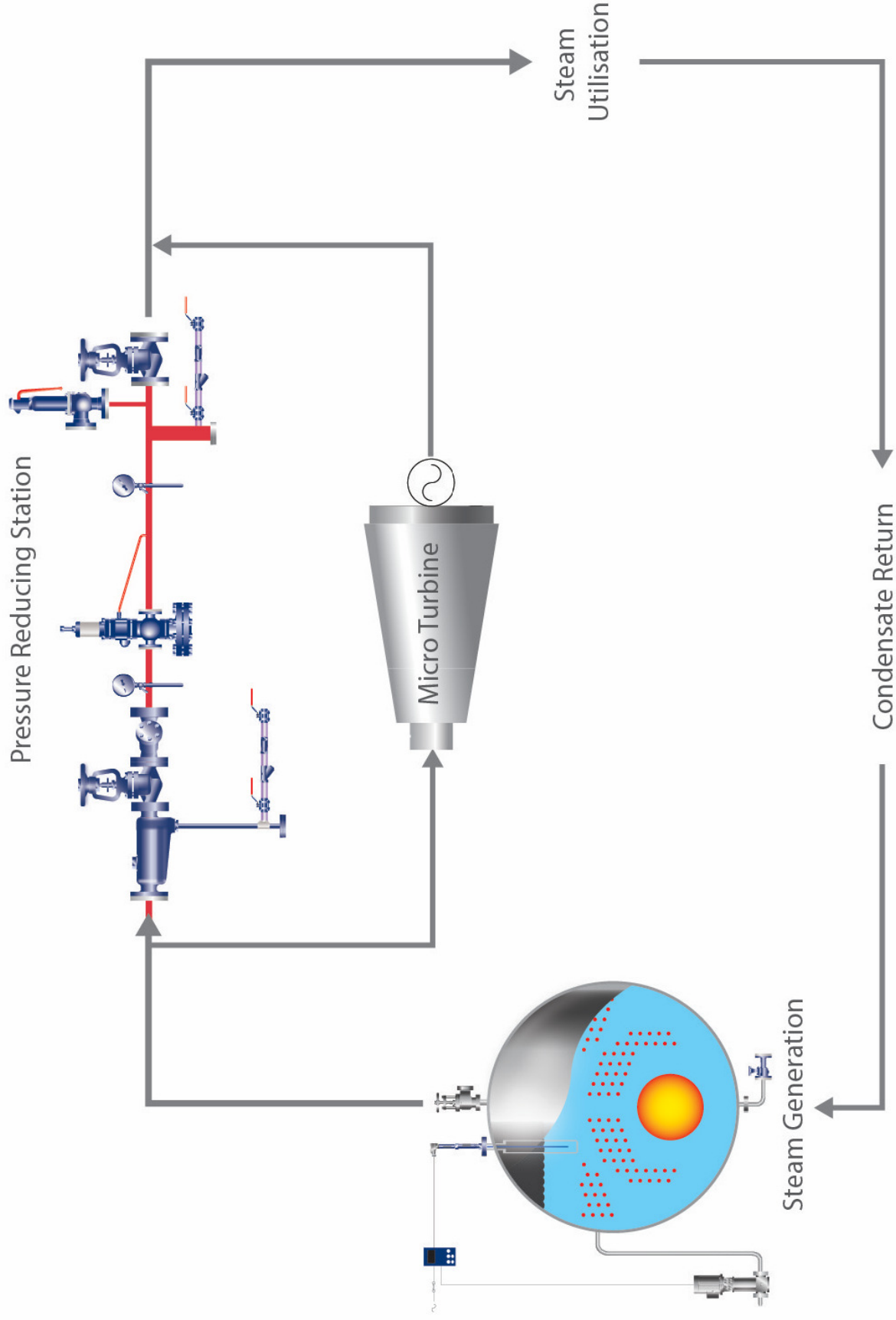
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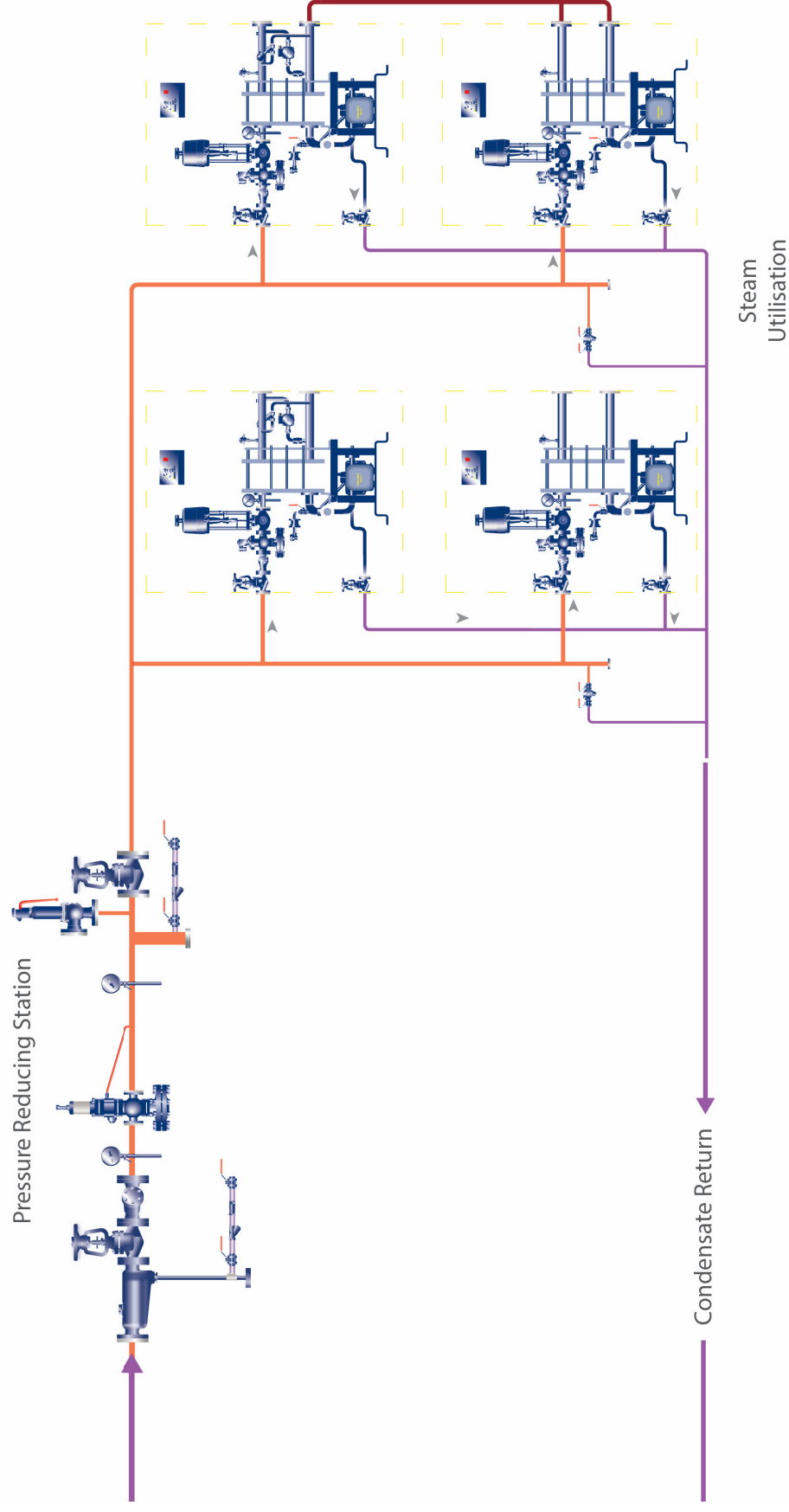


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# The low carbon plantroom

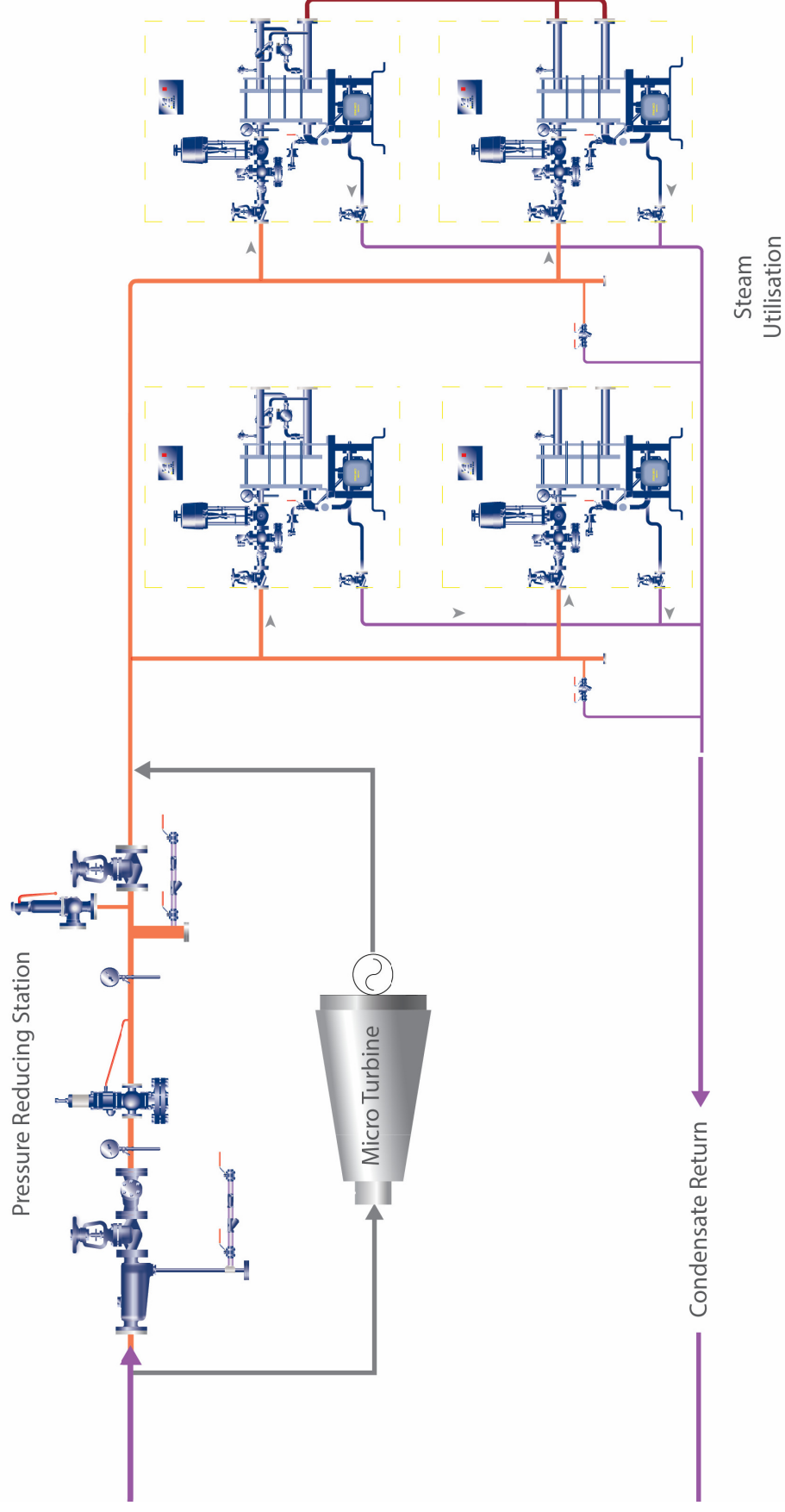


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# The low carbon steam plantroom with power generation



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Thank you for listening

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